

## 13. Static Equilibrium

Tolerances for optics are very tight. We need to support them so they are accurately located.

If forces are applied, we want to determine:

Motion

Distortion

In order to do this, we need to evaluate the system, including the applied forces and the reaction forces.

In this section, we define forces and moments, develop the free body diagram, and use the equations of static equilibrium to solve for reaction forces and moments.

Forces are vectors:

They have a magnitude and direction.

What does a force do?

Can accelerate an object  $F = m a$

Can stretch a spring scale



Forces can be applied:

Units of Pounds on Newtons

1 pound ( $\text{lb}_F$ ) = 4.45 N : 1 N = 0.22 lb

Or they can come from gravity

$W = m g$  ( $g = 9.8 \text{ m/s}^2 = 386 \text{ in/s}^2$ )

1 kg has weight of 9.8 N or 2.2 lbs

1  $\text{lb}_M$  is the mass that weighs 1 pound

1 slug weighs 32.2 lbs

The moment is defined as

$$\begin{aligned}\vec{M}_A &= \vec{r}_{AB} \times \vec{F}_B \\ &= r_{AB} F_B \sin \theta \\ &= r_{AB} \cdot F_{\perp} \\ &= r_{\perp} \cdot F_B\end{aligned}$$

Also called “torque”

Units are in-lb or N-m

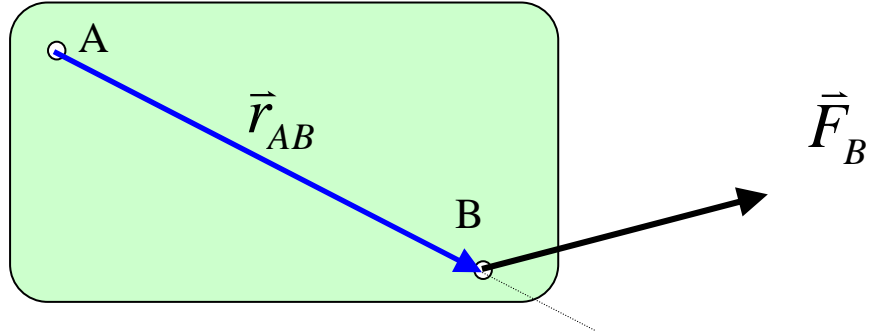
$$1 \text{ N-m} = 8.84 \text{ in-Lb}$$

Moments are “twisting forces”. They make things rotate

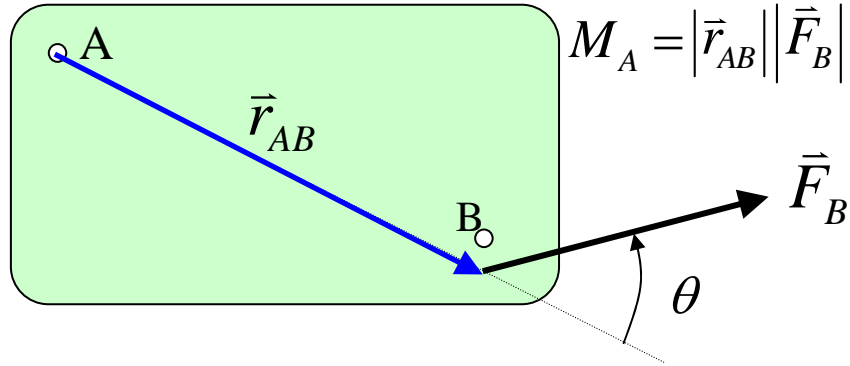


# Defining moment from applied force

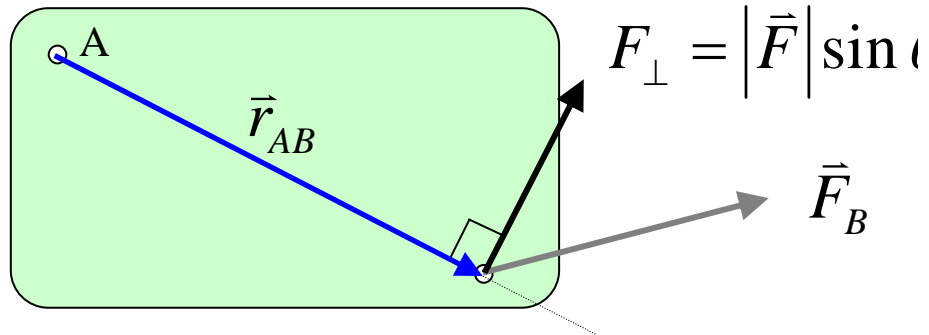
$$\vec{M}_A = \vec{r}_{AB} \times \vec{F}_B$$



$$M_A = |\vec{r}_{AB}| |\vec{F}_B| \sin \theta$$



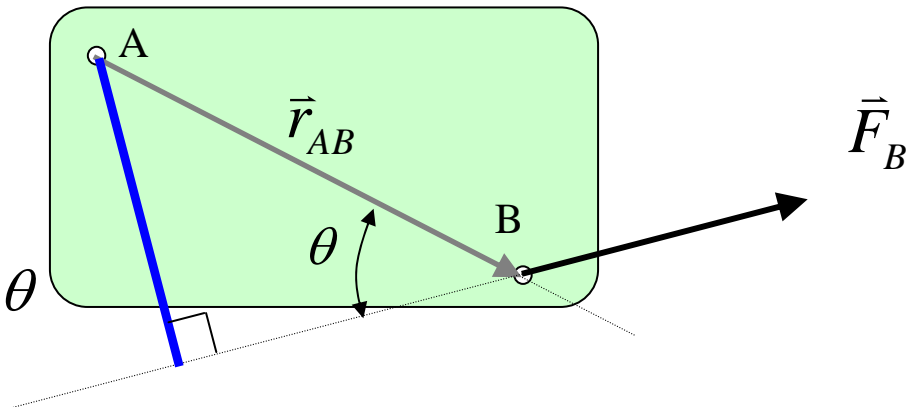
$$M_A = |\vec{r}_{AB}| \cdot F_{\perp}$$



$$F_{\perp} = |\vec{F}| \sin \theta$$

$$M_A = r_{\perp} \cdot |F|$$

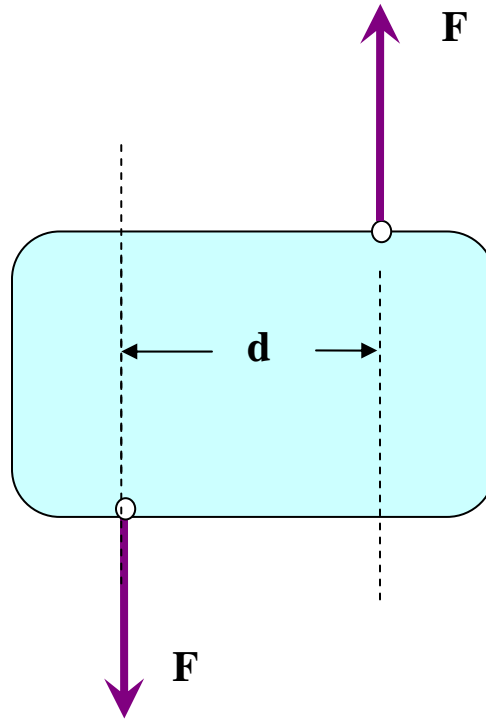
$$r_{\perp} = |\vec{r}_{AB}| \sin \theta$$



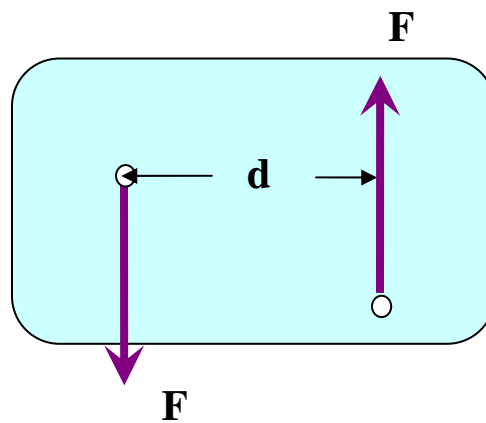
# Force couples

Two forces, equal and opposite in direction, which do not act in the same line cause a pure moment

$$\mathbf{M} = \mathbf{F} \mathbf{d}$$



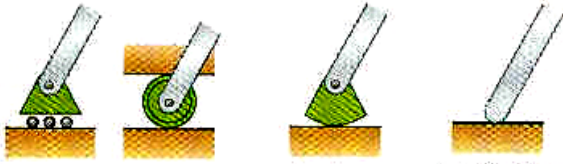



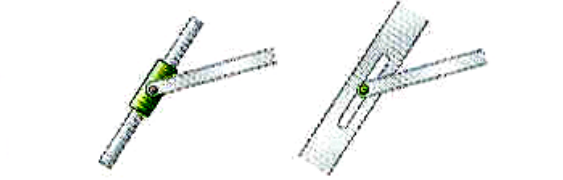
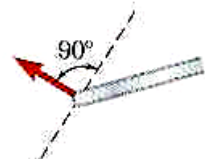

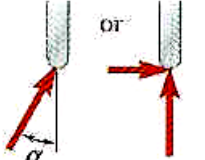
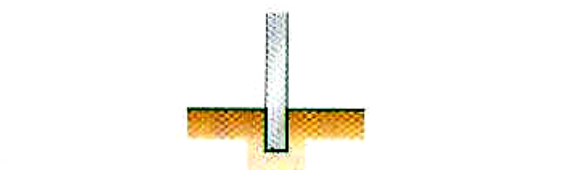
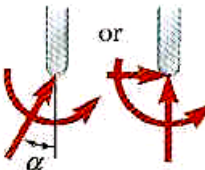
$$\mathbf{M} = \mathbf{F} \mathbf{d}$$



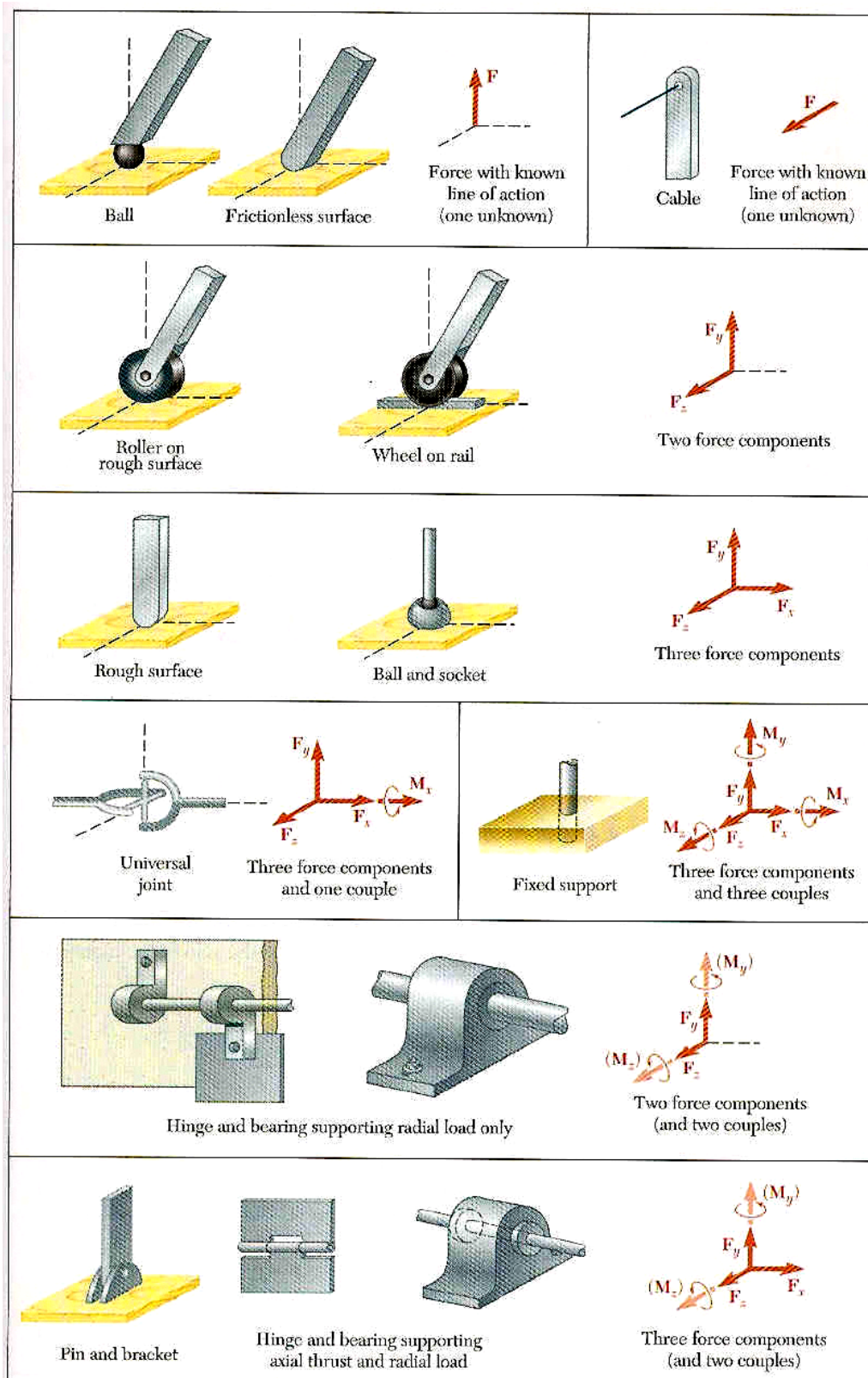
## Constraints

Constraints are attachment points that will maintain their position.

### Idealization of 2D supports

Support or Connection	Reaction	Number of Unknowns
 <p style="text-align: center;">Rollers      Rocker      Frictionless surface</p>	 <p style="text-align: center;">Force with known line of action</p>	1
 <p style="text-align: center;">Short cable      Short link</p>	 <p style="text-align: center;">Force with known line of action</p>	1
 <p style="text-align: center;">Collar on frictionless rod      Frictionless pin in slot</p>	 <p style="text-align: center;">Force with known line of action</p>	1
 <p style="text-align: center;">Frictionless pin or hinge      Rough surface</p>	 <p style="text-align: center;">Force of unknown direction</p>	2
 <p style="text-align: center;">Fixed support</p>	 <p style="text-align: center;">Force and couple</p>	3

# Idealization of 3D supports



## Simple cases

### **Cable**

Can only transmit tension  
along direction of cable

No compression

No moment

No lateral force



## Free Body Diagrams

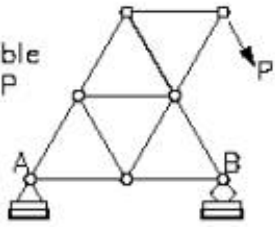
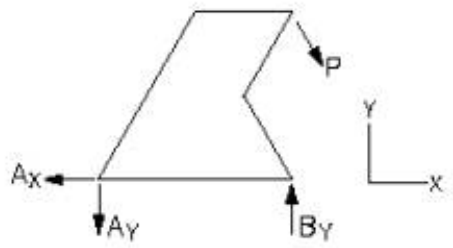
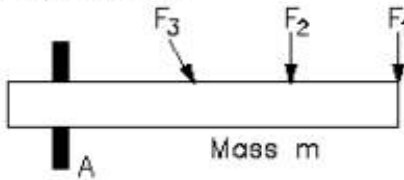
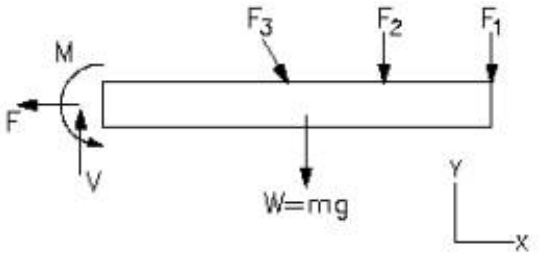
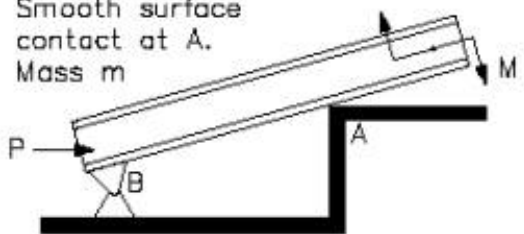
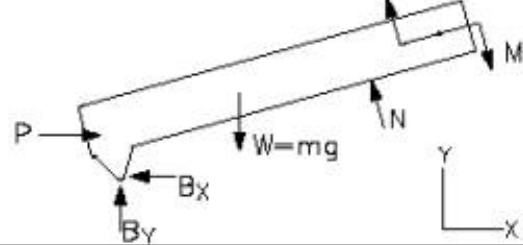
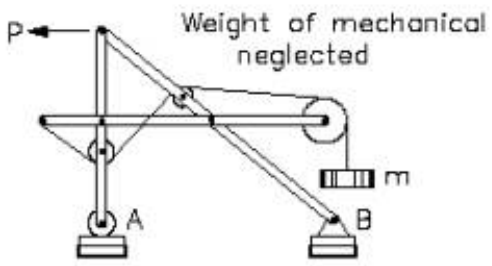
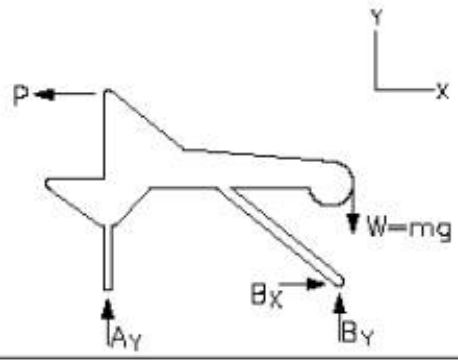
**Step 1. Determine which body or combination of bodies is to be isolated.** The body chosen will usually involve one or more of the desired unknown quantities.

**Step 2.** Next, **isolate the body** or combination of bodies chosen with a diagram that represents its complete external boundaries.

**Step 3. Represent all forces that act on the isolated body** as applied by the removed contacting bodies in their proper positions in the diagram of the isolated body. Do not show the forces that the object exerts on anything else, since these forces do not affect the object itself.

**Step 4. Indicate the choice of coordinate axes** directly on the diagram. Pertinent dimensions may also be represented for convenience. Note, however, that the free-body diagram serves the purpose of focusing accurate attention on the action of the external forces; therefore, the diagram should not be cluttered with excessive information. Force arrows should be clearly distinguished from other arrows to avoid confusion.

When these steps are completed a correct free-body diagram will result. Now, the appropriate equations of equilibrium may be utilized to find the proper solution.

SAMPLE FREE-BODY DIAGRAMS	
Mechanical System	Free-Body Diagram of Isolated Body
<p>1. Plane truss</p> <p>Weight of truss assumed negligible compared with <math>P</math></p> 	
<p>2. Cantilever beam</p> 	
<p>3. Beam</p> <p>Smooth surface contact at A.</p> <p>Mass <math>m</math></p> 	
<p>4. Rigid system of interconnected bodies analyzed as a single unit</p> <p>Weight of mechanical neglected</p> 	

For a rigid body to be static, the net sum of forces and moments acting on it must be zero.

$$\sum \vec{F} = 0$$

$$\sum \vec{M} = 0$$

$$\sum F_x = 0$$

$$\sum M_x = 0$$

$$\sum F_y = 0$$

$$\sum M_y = 0$$

$$\sum F_z = 0$$

$$\sum M_z = 0$$

In general six equations, in the plane this reduces to 3

$$\sum F_x = 0$$

$$\sum F_y = 0$$

$$\sum M = 0$$

# Solving Statics problems

Determine reaction forces for static equilibrium.

## 1. Draw Free Body Diagram

Decide if the problem is solvable

a. How many unknowns?

b. How many equations can you write?

## 2. Write equations to sum forces and moments to be 0

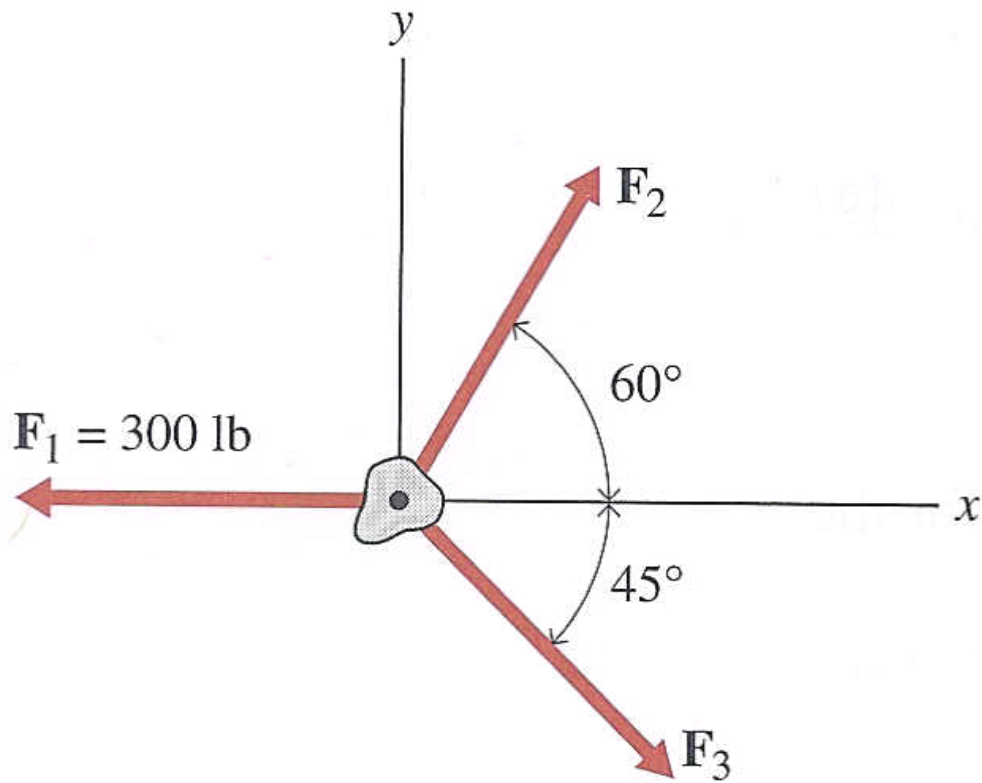
a. Use reaction forces as unknowns

b. Be smart about coordinates and choice of points for summing moments

## 3. Solve equations for reaction forces

## 4. Check your answer and the direction

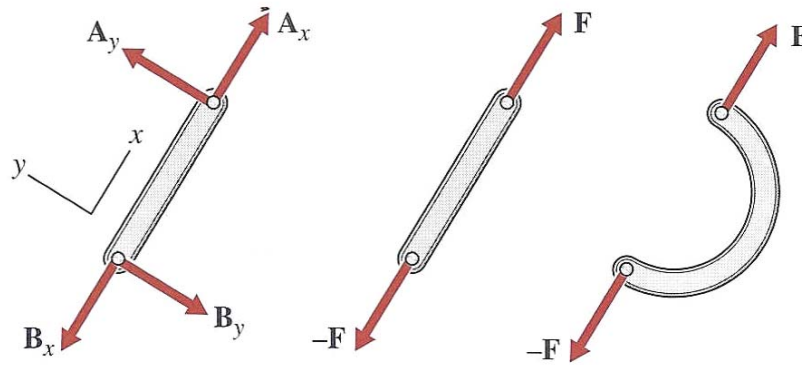
## 2D Particle Example



- Determine magnitude of  $F_2$  and  $F_3$

## Link Pin joint at both ends

Equilibrium requires that the forces be equal, opposite and collinear.



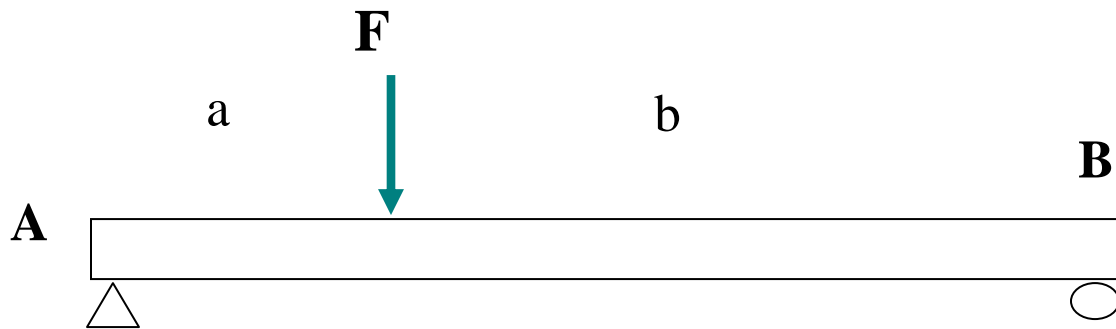
Therefore, for this member  $A_y = B_y = 0$

Pin joint will not transmit a moment

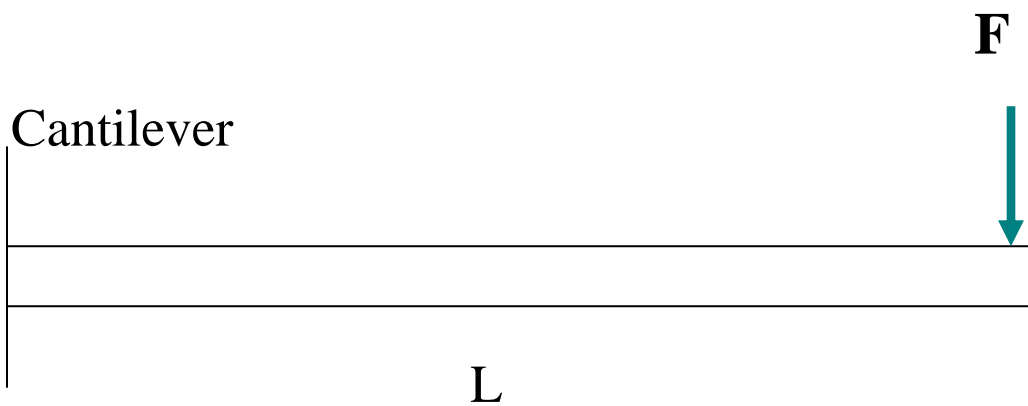
## Simple Examples

Determine reaction forces and moments:

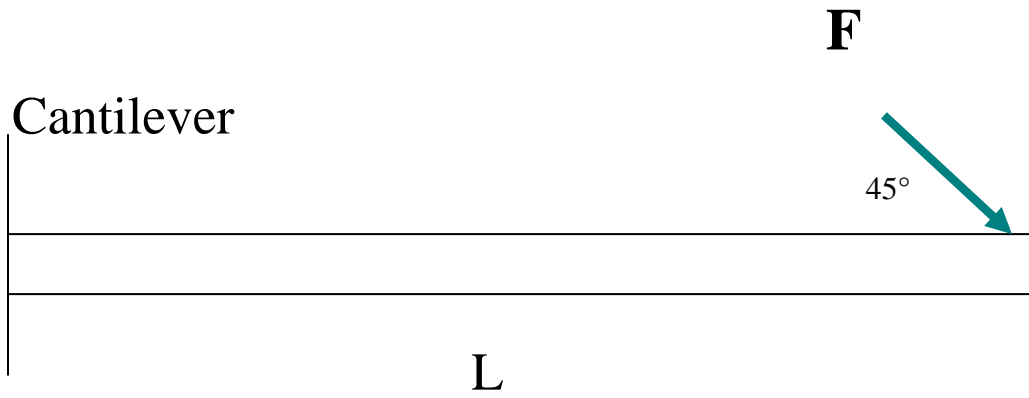
Simple support



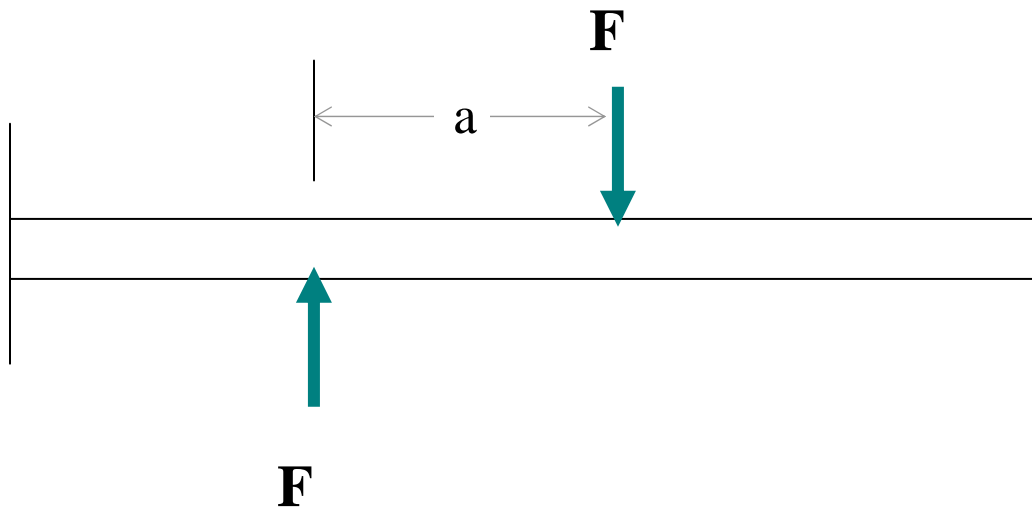
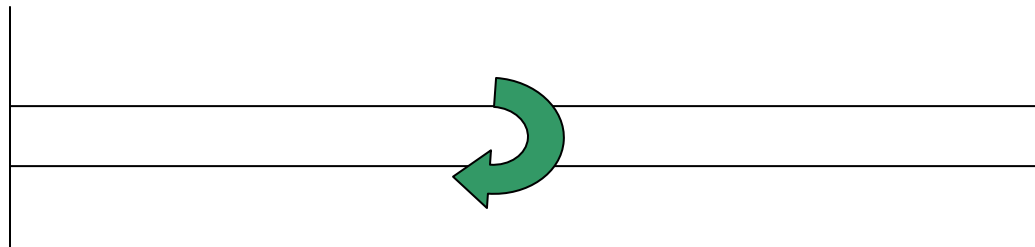
Cantilever

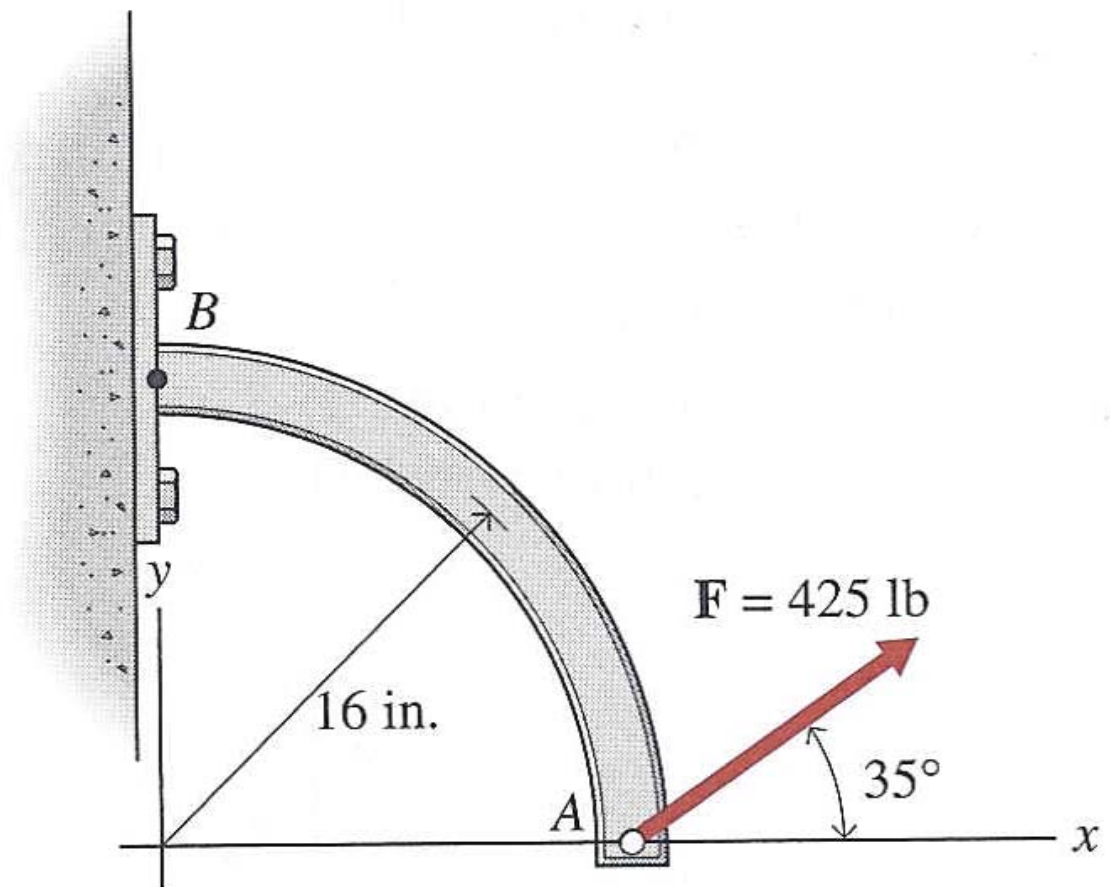


# X and Y components

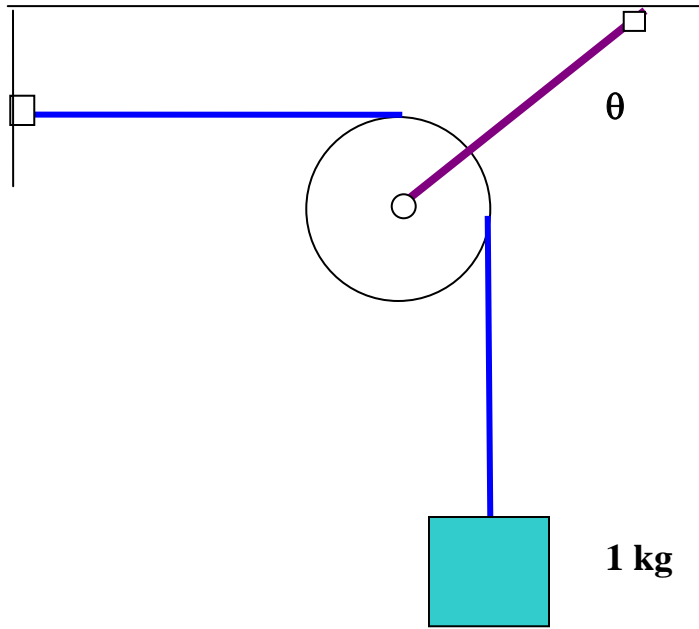


# Reaction from moments

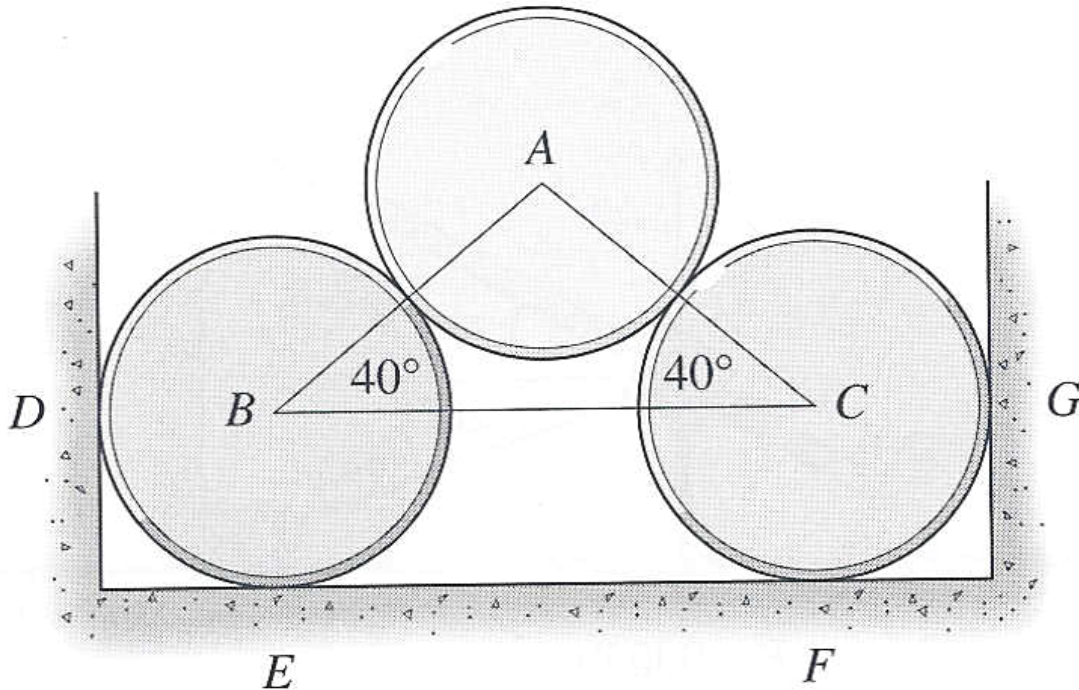




Example: Hanging a mass, using a pulley



## 2D Cylinder Example



### **Specifications:**

- Cylinder diameter = 250 mm
- Cylinder mass = 245 kg

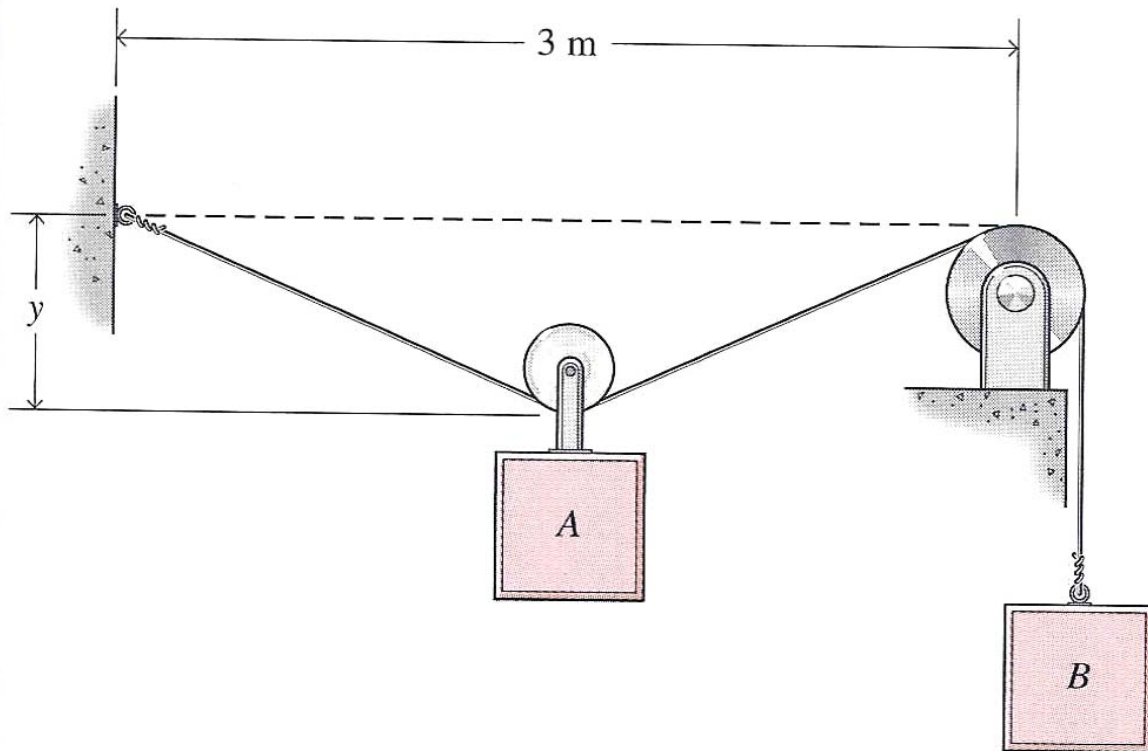
### **Assumptions:**

- Smooth homogeneous cylinders

### **Determine:**

- Force of cylinder B on cylinder A
- Forces at D and E on cylinder B

## 2D Pulley Example



### **Specifications:**

- Mass of block A = 22 kg
- Mass of block B = 34 kg

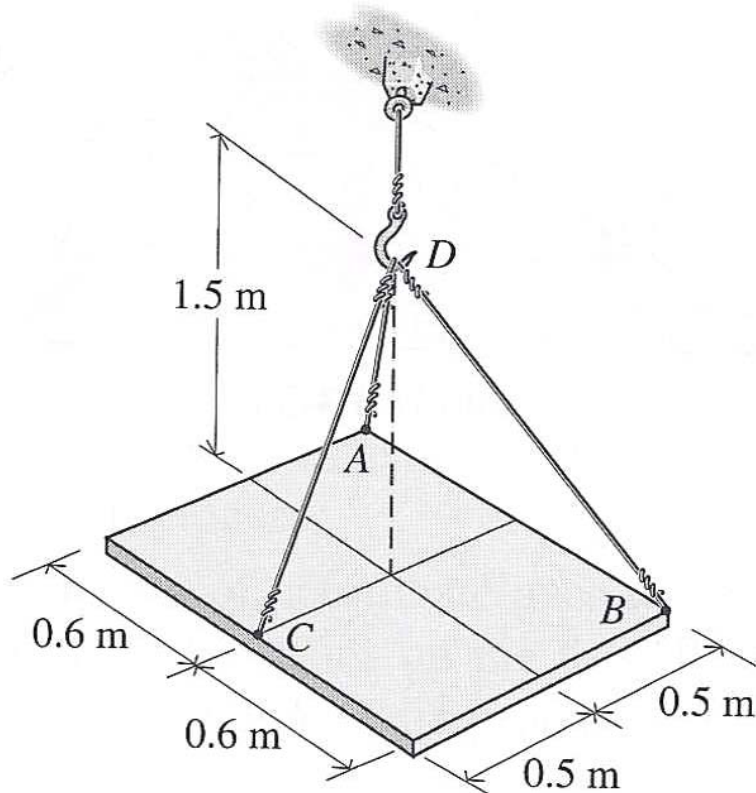
### **Assumptions:**

- Pulleys are frictionless
- Block A is free to roll
- Cable system is continuous

### **Determine:**

- Displacement “y” for equilibrium

## 3D Cable System Example



### **Specifications:**

- Weight of plate = 250 lb

### **Assumptions:**

- Plate is homogeneous

### **Determine:**

- Force in each supporting cable

## Overconstraint

**Each body has a total of 6 degrees of freedom that define its position**

**Such as  $x, y, z, \theta_x, \theta_y, \theta_z$**

**These lead to 6 Equations that can be used to solve for reaction forces:**

$$\sum F_x = 0$$

$$\sum F_y = 0$$

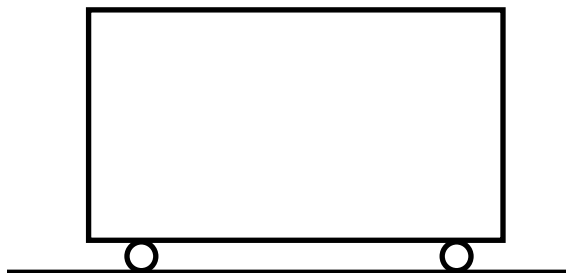
$$\sum F_z = 0$$

$$\sum M_x = 0$$

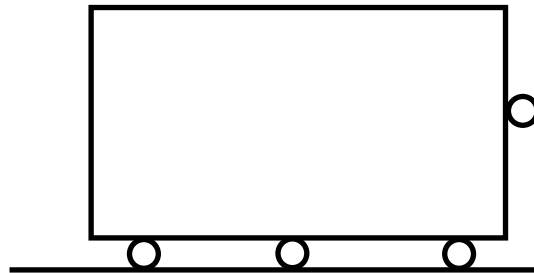
$$\sum M_y = 0$$

$$\sum M_z = 0$$

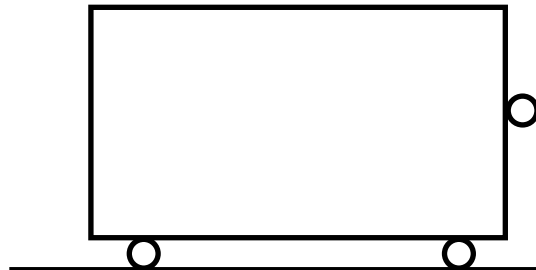
**If the mechanical constraints provide an attachment so that one or more degrees of freedom are free, the body is underconstrained**



**If the mechanical constraints provide an attachment so that there is no unique solution for the reaction forces, the body is overconstrained**



**A body that is neither overconstrained nor underconstrained is called static determinant**



**Static equations must have 6 unknowns for 3-space, or 3 unknowns for in-plane**

**If you are not sure, then try solving for the reaction forces and moments.**

**If you have a unique solution  
static determinant**

**If you have multiple solutions (more unknowns  
than equations)**

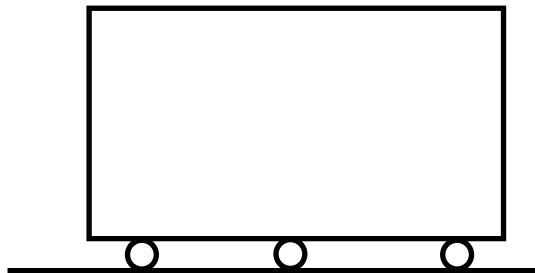
**Overconstrained  
reaction forces can be pushing against each  
other**

**If you have more equations than unknowns**

**Underconstrained  
Some degree of freedom is not constrained  
and could move**

**Try to figure out what degree of freedom has not  
been constrained.**

**You can be overconstrained and  
underconstrained at the same time!**



## Analysis of Structures

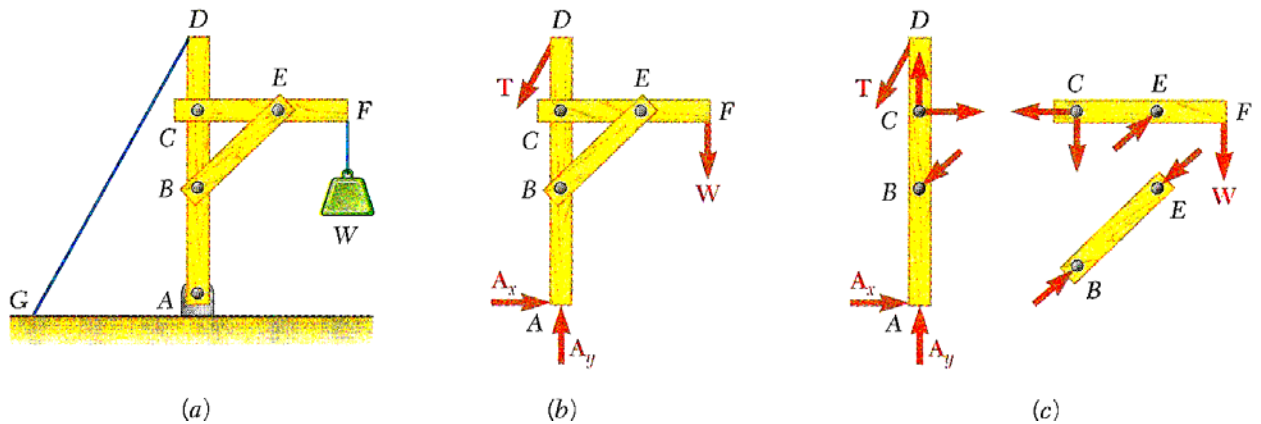


Figure (a) – Crane example

Figure (b) – Free body diagram of crane showing external forces.

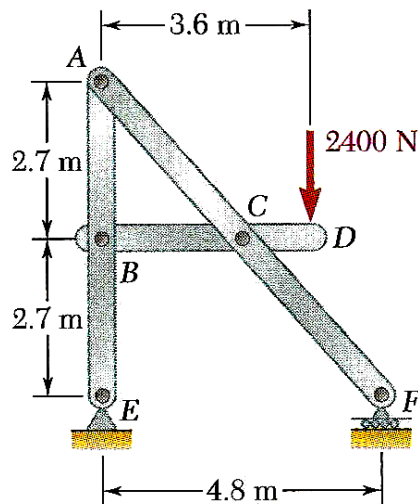
Figure (c) – Dismembered crane showing member forces. From the point of view of the structure as a whole, these forces are considered to be internal forces.

The internal forces conform to Newton's third law – the forces of action and reaction between bodies in contact have the same magnitude, same line of action and opposite sense.

When structures, like the one shown above, contain members other than two force members, they are considered to be frames or machines. Typically, frames are rigid structures and machines are not.

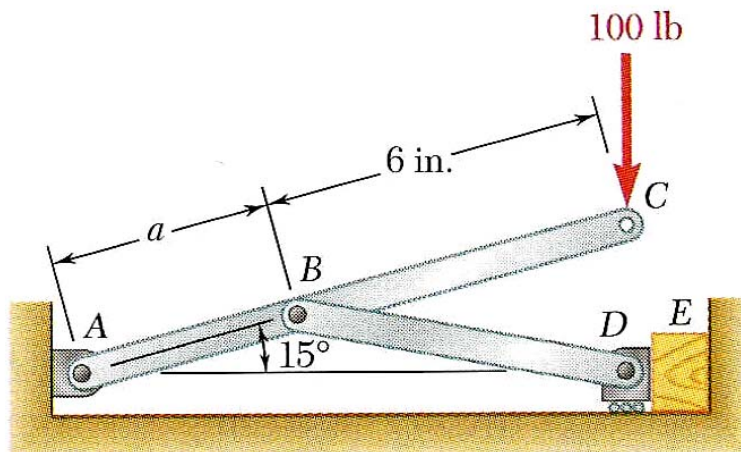
## Frames

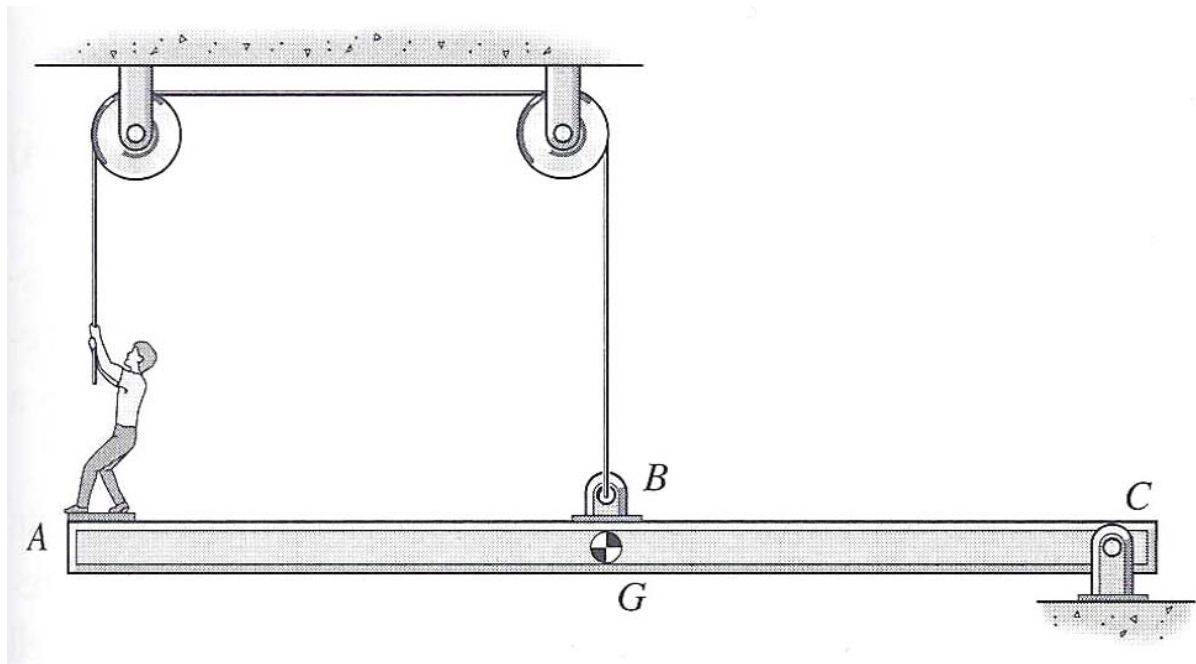
- Designed to support loads.
- Typically rigid, stationary and fully constrained.
- Contains at least one multi-force member.



## Machines

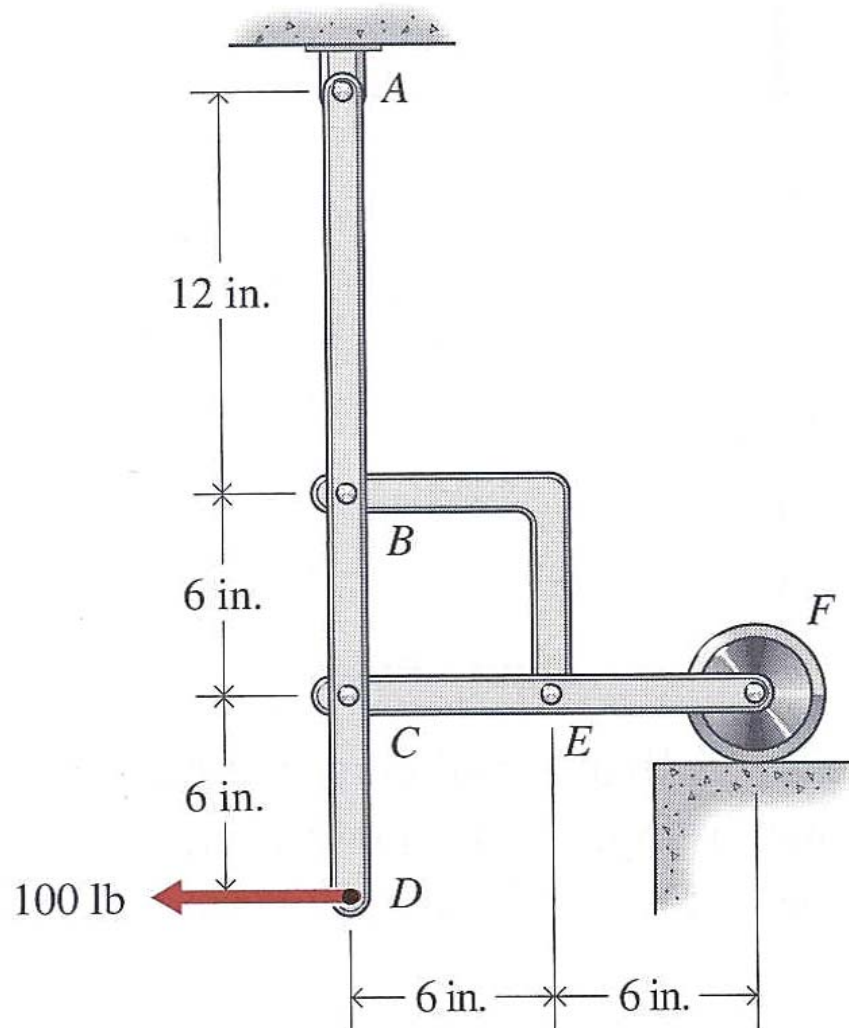
- Designed to transmit or modify forces.
- Contain moving parts.
- Contains at least one multi-force member.





Assuming the beam does not fall, what is the direction of the force applied to the beam at C?

## Example



Determine the forces acting on member ABCD.