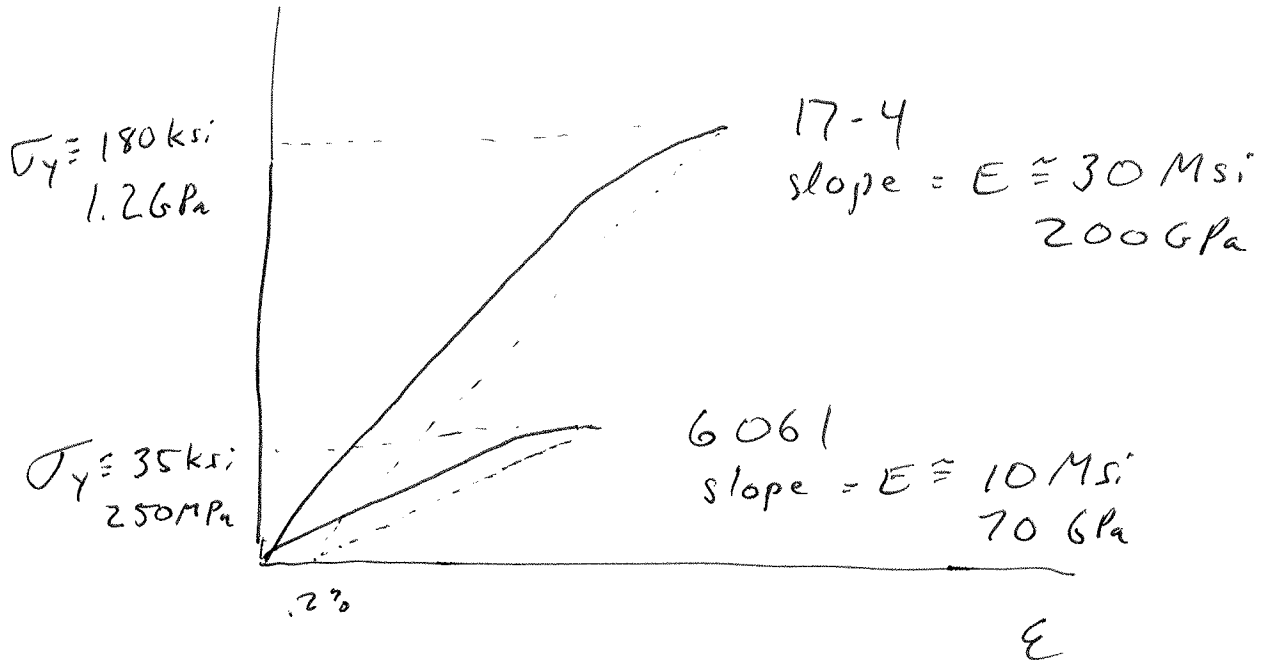


Optical Engineering 421/521 – Fall 2009

Midterm 2 50 minutes, closed book, closed notes, no calculators

November 16, 2009

- 1.) (10) Draw a plot showing the relationship between stress and strain for 6061 aluminum. Label the axes and give units. On the same plot, provide an approximate curve for 17-4 stainless steel (spring steel). Indicate Young's modulus and the yield strength for both.



- 2.) (6) Name a metal with exceptionally high specific stiffness. Compare Young's modulus and density for this material with aluminum.

beryllium

aluminum is

E

42 msi
290 GPa

$$\frac{E_{Be}}{E_{Al}} \approx 4$$

10 msi
68 GPa (~70)

ρ

.067 lb/in³
1.85 g/cm³

$$\frac{\rho_{Be}}{\rho_{Al}} \approx \frac{2}{3}$$

0.1 lb/in³
2.7 g/cm³

- 3.) (5) Provide the relationship that describes the change in focal length for a mirror due to a change in temperature.

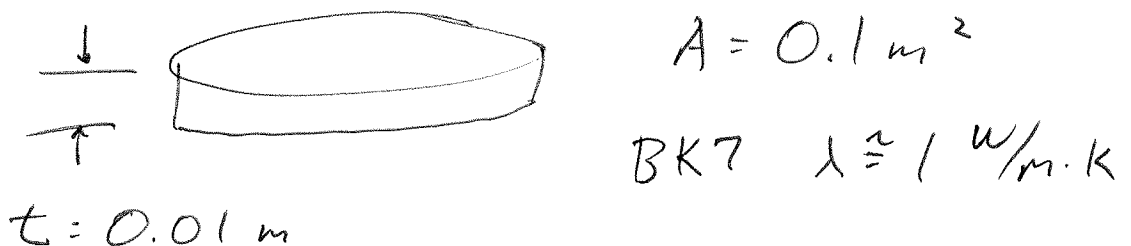
All dimensions scale with CTE α

$$\Delta f = f \cdot \alpha \Delta T$$

- 4.) (6) List the three basic steps to create a solid model of a cube with a threaded hole in it using Solid Works..

1. Sketch
2. Extrude
3. Use the hole wizard

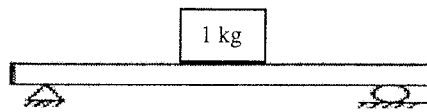
- 5.) (5) One side of a 1 cm thick BK7 window is 1°C warmer than the other. If the window has 0.1 m² cross sectional area, calculate the power in Watts conducted by the glass.



$$\text{Power} = \frac{\lambda A \Delta T}{t} = \frac{1 (.1)(1)}{.01}$$

$$= 10 \text{ Watts}$$

6) (15) A 1 kg mass is placed in the center of a beam, causing 0.1 mm deflection. Calculate the resonant frequency (ignoring the weight of the beam). Assuming 2% damping, sketch a plot of the transmissibility T (ratio of motion at the center of the beam to base motion) as a function of frequency for this system. Label the axes and show the values of T for low frequency and at resonance.



$$g = 9.8 \text{ m/s}^2 \approx 10 \text{ m/s}^2$$

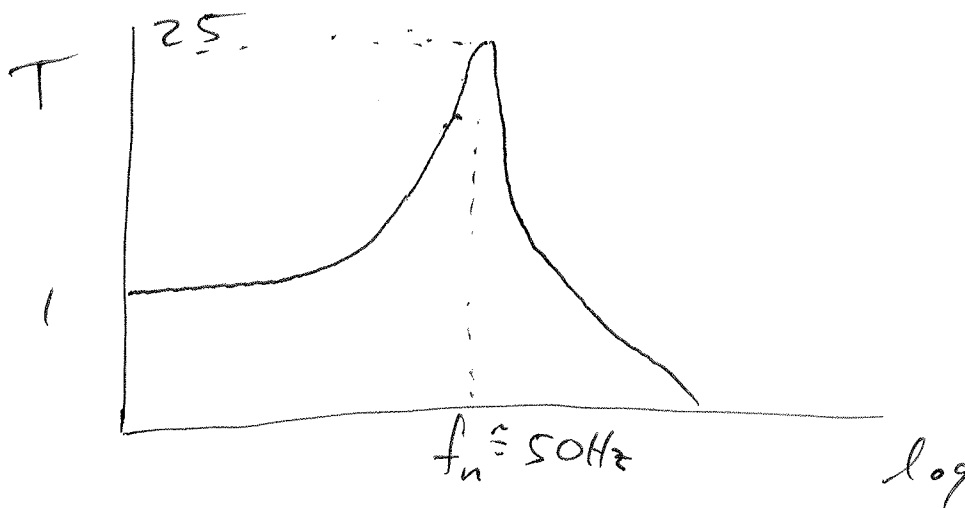


$$\omega = \sqrt{k/m}$$

$$\delta = F/k = \frac{mg}{k} \quad \frac{k}{m} = \frac{g}{\delta}$$

$$\omega = \sqrt{\frac{g}{\delta}} = \sqrt{\frac{10 \text{ m/s}^2}{10^{-4}}} = \sqrt{10^5} \approx 316 \text{ rad/s}$$

$$Q = \frac{1}{2\zeta} = \frac{1}{.04} = 25 \quad \approx 50 \text{ Hz}$$



7) (25) A BK7 beamsplitter cube is bonded to aluminum with 0.2 mm thick RTV elastomeric adhesive. The bond area is circular, 11 mm diameter. (~100 mm² area)

The adhesive has

G 1 MPa shear modulus

E_b 1 GPa bulk modulus

σ_u 5 MPa adhesive strength, shear and normal

$$t = .2 \text{ mm}$$

$$A = 100 \text{ mm}^2$$

- Calculate the strength of the bond for shear loading. (The shear force in N required to fail the adhesive.)
- Calculate the lateral (shear) stiffness of the bond. (The force in N per mm lateral displacement.)
- Calculate the normal (compressive) stiffness of the bond. (The force in N per mm axial displacement.)
- Calculate the maximum adhesive shear strain for 11°C change in temperature.
- Calculate the maximum shear stress in the adhesive for the 11°C temperature change.

$$a. \sigma_u = \frac{F_u}{A} = 5 \text{ MPa} = 5 \text{ N/mm}^2, \quad A = 100 \text{ mm}^2$$

$$F = 500 \text{ N}$$

$$\rightarrow b. k_s = \frac{V}{\delta_s} \quad \delta_s = \frac{V \cdot t}{G \cdot A} \quad k_s = \frac{G \cdot A}{t} = \frac{1 \text{ N/mm}^2 \cdot 100 \text{ mm}^2}{0.2 \text{ mm}}$$

$$k_s = 500 \text{ N/mm}$$

$$\downarrow c. k_N = \frac{F}{\delta_N} \quad \delta_N = \frac{F \cdot t}{E_b \cdot A} \quad k_N = \frac{E_b \cdot A}{t}$$

$$k_N = 500,000 \text{ N/mm}$$

$$d. \text{ Al. } \alpha = 23 \text{ ppm/}^\circ\text{C} \quad \Delta \alpha = 16 \text{ ppm/}^\circ\text{C}$$

$$\text{BK7 } \alpha = 7 \text{ ppm/}^\circ\text{C}$$

$$\Delta a = \left(\frac{11}{2}\right) \cdot 16 \text{ ppm/}^\circ\text{C} \cdot 11^\circ$$

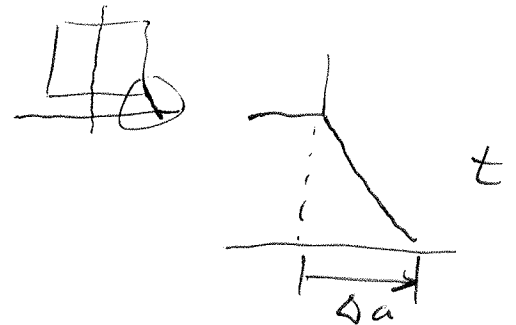
$$= 11^2 \cdot 8 \text{ } \mu\text{m}$$

$$\approx 121 \cdot 8 = 1000 \text{ } \mu\text{m}$$

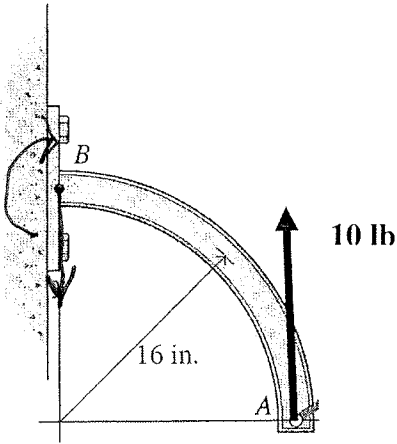
$$= 1 \text{ mm}$$

$$\gamma = \frac{1 \text{ } \mu\text{m}}{200 \text{ } \mu\text{m}} = .005$$

$$\tau = G \cdot \gamma = 0.005 \text{ MPa}$$



8) (5) Calculate the reactions at B for static equilibrium when a 10 lb vertical force is applied at A.



$$F_{By} = 10 \text{ lb}$$

$$F_{Bx} = 0$$

$$M_B = 10 \text{ lb} \cdot 16'' = 160 \text{ in} \cdot \text{lb}$$

9) (5) What determines the strength of glass? *Critical flaw size*
 How can you determine allowable stresses for glass, assuming a standard ground finish?

For ground surface, this is statistical.

Use Weibull statistics to relate probability of failure to tensile stress at surface

$$P = 1 - e^{-\left(\frac{\sigma}{\sigma_0}\right)^m}$$

σ = characteristic strength
 m = Weibull modulus

10) (5) What is a Belleville washer? Why are these used?

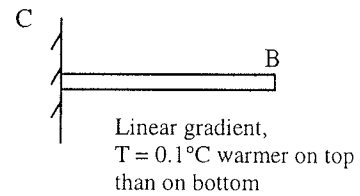
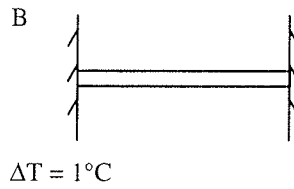
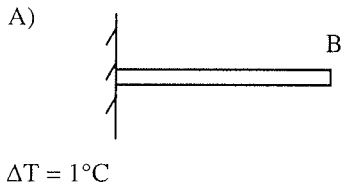
Conic shaped washer.



Used as spring for assembly

11 (13) Consider a bar made of aluminum 10 cm long, 1 cm x 1 cm cross section. Determine the following:

- A) Motion of point B if the bar is heated 1°C, and allowed to expand
- B) stress in the material if the bar is heated 1°C while constrained.
- C) motion of point B for the case where a thermal gradient is applied, with the top of the bar 0.1°C warmer than the bottom.



$$\alpha = 23 \text{ ppm}/^\circ\text{C}$$

$$l = 100 \text{ mm}$$

$$E = 70 \text{ GPa} = 70,000 \text{ N/mm}^2$$

$$A = 100 \text{ mm}^2$$

a) $\Delta l = l \alpha \Delta T = 2300 \mu\text{m} = 2.3 \text{ mm} \rightarrow$

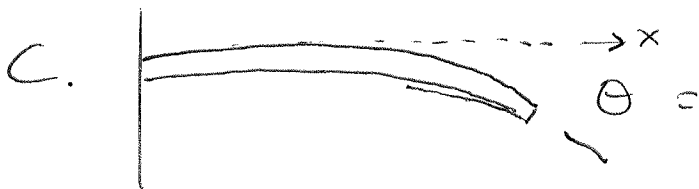
b) Super position $\Delta l = \frac{Fl}{EA} = \frac{\sigma l}{E}$

$$\sigma = \frac{\Delta l}{l} \cdot E = E \alpha \Delta T$$

$$= 70 \text{ GPa} \cdot 23 \text{ ppm}$$

$$= 10 \text{ MPa} \cdot 23 \text{ ppm}$$

$$= 230 \text{ psi}$$



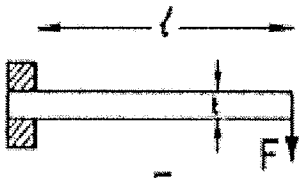

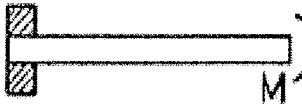



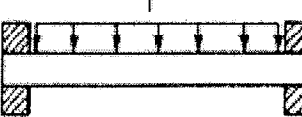
$$\theta = \frac{\alpha L \Delta T}{t} = \frac{23 \text{ ppm} \cdot 100 \text{ mm} \cdot 1^\circ\text{C}}{10 \text{ mm}}$$

$$= 230 \mu\text{rad}$$

$$\theta(x) = \frac{\alpha \Delta T}{t} \cdot x + \theta^0$$

$$\delta(x) = \int \theta dx = \frac{1}{2} \frac{\alpha \Delta T}{t} x^2 + \theta^0 = \frac{230 \mu\text{rad}}{2} \cdot 100 \text{ mm}$$

$$= 115 \mu\text{rad} \cdot 100 \text{ mm} = 11.5 \mu\text{m}$$

	C_1	C_2
	3	2
	8	6
	2	1
	48	16
	$\frac{384}{5}$	24
	192	—
	384	—

$$\delta = \frac{Fl^3}{C_1EI} = \frac{Ml^2}{C_1EI}$$

$$\theta = \frac{Fl^2}{C_2EI} = \frac{Ml}{C_2EI}$$

- E = YOUNGS MODULUS (N/m²)
- δ = DEFLECTION (m)
- F = FORCE (N)
- M = MOMENT (Nm)
- l = LENGTH (m)
- b = WIDTH (m)
- t = DEPTH (m)
- θ = END SLOPE (-)
- I = SEE TABLE 2 (m⁴)